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STATUS REPORT

RESEARCH FOR DETERMINATION OF BLAST  
EFFECTS ON RE-ENTRY VEHICLE STRUCTURE

LAWRENCE ROSENDORF

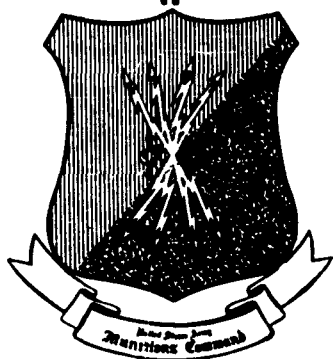
IRVING KINTISH

MARCH 1963

PICATINNY ARSENAL  
DOVER, NEW JERSEY

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Technical Memorandum 1152

STATUS REPORT -

RESEARCH FOR DETERMINATION OF BLAST EFFECTS  
ON RE-ENTRY VEHICLE STRUCTURES

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March 1963

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# ABSTRACT

This report outlines the experimental work performed by Picatinny Arsenal during the period 1 July 1962 to 1 January 1963 to derive by model testing, analytical and experimental techniques for the determination of blast loading and target response of ballistic missile re-entry structures. This information is considered a prerequisite in determining the kill and vulnerability of such targets to nuclear attack. Simple shell structures, of varying geometric parameters, have been instrumented with pressure transducers and strain gages and subjected to air blast from varying H. E. charge weights. A preliminary analytical approach to predict the incipient buckling pressure of the targets (cylindrical portion only), has been developed. This approach is based on a "dynamic load factor" which enables one to compute the dynamic buckling pressure from the static formulations. Plans are discussed for multi-ton H. E. tests.



## Introduction

This report is submitted to the Flight Dynamics Laboratory, Aeronautical Systems Division, Wright Patterson Air Force Base for work conducted under Air Force MIPR 33 (657) -2-R&D-233. Past work has been jointly funded by ASD and Picatinny Arsenal, and is now being continued as part of the DASA WEB blast program; sub-task 01.019, Elast Effects on Re-entry Vehicles. The experimental work on this phase of the 01.019 program is the responsibility of the Ballistic Research Laboratories and Picatinny Arsenal.

This report presents the results of Picatinny Arsenal's efforts on the Elast Effects on Re-entry Vehicles program during the period July 1962 to January 1963. The results of the initial work are reported in Picatinny Arsenal Technical Memorandum Number 114-62 (Reference 1).

## Objectives

The basic objective of this program is to derive experimental and analytical techniques for determination of blast loading and target response of ballistic missile re-entry vehicles in order to determine the conditions leading to kill of these targets. The course of action currently planned by BRL-PA is primarily that of a systematic progressive development of information on the response of missile bodies, beginning with deformation by impulsive loading on stationary targets and continuing to consider all possible factors in the actual dynamic re-entry situation.

The objective of the initial experimental phase of the PA effort is limited to determination of pressure loading and structural response of simple RV geometrical shapes (viz. cylinders, cones, and hemispheres) for extrapolation to the impulsive loading effects characteristic of duration times of nuclear detonations. The preliminary work is limited to the use of stationary targets in order to obtain the basic data required before going on to the dynamic situation of actual re-entry vehicles. The approach is to measure the surface pressure loads and structural deformation resulting from chemical HE bursts over a wide range of HE charge weight. Data points will then be obtained for as wide a spread of impulse duration times as possible. Attempts would then follow to extrapolate the data and develop analytical expressions for target response that will be valid in the higher impulse duration region, characteristic of nuclear detonations. It will then be desirable to test these predictions by subjecting models to full scale nuclear tests and adjust the theory as required.

### Current Test Program

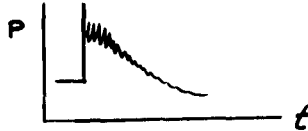
In the current experimental portion of the blast program at Picatinny Arsenal, models are mounted on test stands and subjected to air blast loadings from chemical high explosives. The tests at Picatinny are limited to 50 lbs. of high explosives. The models being tested are a combination of truncated cone and circular cylinder with hemispherical end caps, to include the basic geometrical shapes of interest for re-entry vehicles. The present models have two skin thicknesses, 1/8 inch and 1/16 inch. The cylindrical portions of each model are 7-1/2 inches in diameter and 7-1/2 inches long, and the conical portions have a  $R_2 = 7-1/2$  inches,  $R_1 = 5$  inches, and  $L = 7-1/2$  inches. The total length of each model is 20 inches. The models are fabricated of aluminum and are reinforced by 3/8 inch aluminum bulkheads separating the cylindrical and conical sections from the end caps. The model having a 1/16 inch wall thickness is noted as target 1 and the model with a 1/8 inch wall thickness is noted as target 2. (See Figs. 1 thru 5). Additional photos of interest, Figs. 6 and 7 are also included.

Tech Memo 114-62 (Reference 1) describes Phase I of the blast program. Entailed are the evaluation of test stands and instrumentation check-out. It was decided that the test stands were to be made as rigid and immovable as was practically possible. Dr. R. Cooper of Aerospace Corp. suggested the need for accelerometers to be mounted on the bulkheads of the models so as to monitor the movement of the model in its supported condition. Time of arrival gages were placed alongside the models so as to determine the free air over pressure.

In phase II of the program the models were instrumented more fully. Atlantic Research Corp., pressure transducers were flush mounted on the skin of the model at 0 degrees and 45 degrees on both the cone and cylinder portions. Baldwin-Lima-Hamilton-SR-4 strain gages were mounted at 0 degrees and 90 degrees on both the cone and cylinder. The models were placed 15 feet from 10 lbs. of composition C-4 explosive. Two 4-beam oscilloscopes coupled with 35 mm framing cameras were used to record the data.

Twelve tests were performed and the data was reduced. The pressure-time records indicated a spike followed by what is believed to be gage-resonance. A sample pressure-time record is sketched below for a 0° Position. Figures 12 and 13 are the actual pressure-time records for the cylindrical and conical portions of

the target for test SP-3 (See Table 1).



This is typical of the records obtained at the face-on position of the cylinder and cone. The peak pressure (spike) has been explained as the Rankine-Hugoniot Peak reflected pressure.

This is computed from the relation,

$$\frac{P_r - P_o}{P_s - P_o} = 2 + \frac{6P_s}{P_s + 7P_o} \quad , \quad \text{where; } P_o = \text{ambient pressure}$$

$$P_s - P_o = \text{Free Air over pressure}$$

$$P_r - P_o = \text{Peak reflected pressure}$$

For  $P_s - P_o = 24.04$  psig at 15 ft. from 10 lbs. C-4 explosive and  $P_o = 14.7$  psia,  $P_r - P_o = 88.8$  psig.

The average of 10 reflected pressure peak spikes is 88.3 psig at the zero degree (face-on) position on the cylinders, which is in excellent agreement with the R-H. predicted value. The average of nine reflected pressure peak readings taken from the cone in a face-on condition was 117 psig.

Since the peak spike pressure occurs almost instantaneously, the positive impulse resulting from it does not contribute significantly to the total impulse that the target sees. Glasstone (Ref. 2) suggests that for large cylindrical structures, the duration of the reflected pressure portion of the pulse can be approximated by  $t_s = \frac{3 \times \text{Radius}}{\text{Shock velocity}}$ . For our specific case this

would lead to  $\frac{3R}{U} = .535$  msec. In the Picatinny Arsenal tests,

the total positive phase duration is approximately 2 msec and as mentioned before, the reflected pressure portion occurs almost instantaneously. Thus, data obtained in this respect does not agree with Glasstone's formulation. As future tests are performed on various geometric targets, this formulation will again be checked.

It should be further pointed out that the data was reduced from the film records by averaging the resonant peaks and omitting the spike for reasons mentioned previously. The data is presented in Table 1.

TABLE 1

Average Peak Reflected Pressure (psig) Omitting  
Rankine-Hugoniot Spike

Shot #	0° Cyl.	45° Cyl.	0° Cone	45° Cone
18	63	54	50	48
19	64	--	43	42
20	73	--	91	47
21	74	50	73	29
22	54	46	83	29
23	57	47	96	29
24	50	30	77	--
28	62	47	90	23
sp3	59	64	69	38
4	54	74	85	53
5	68	60	89	43
6	45	77	53	52
AVG.	60.3	46.7	75	35

All of the data was reduced by the same person and in the same manner so as to eliminate, as much as possible, differences in interpretation of the film records. However, due to the "resonance" that takes place the interpretation is still somewhat vague.

Despite these difficulties two points seem to be of general interest. The first and seemingly obvious point is that in most cases the pressure was higher at zero degrees than at 45 degrees on the cone and cylinder, though individual variations existed. The second point that was observed is that the average 0° pressures on the cone were higher than the corresponding pressures on the cylinder while the average 45° pressures on the cone were lower. Here again individual variations existed and the data is not yet felt to be adequately reproducible. Further study is needed in this area to determine the diffractive pattern of the shock wave over the cone. It is felt that the use of a shock tube could prove extremely valuable.

Several possible sources of error have existed in our experiments. The most probable and significant of these is the fact that the composition C-4 explosive was hand molded at the test site. Variations in symmetry and density of explosive undoubtedly existed and could account for some part of the deviation noted. It is suspected that "end effects" existed and created a non-hemispherical shock wave.

In the initial experiments a surveyor's transit was used to align the explosive and target. As subsequent experiments were carried out, the craters formed by the explosive had to be refilled. The elevation of the explosive charge used in these subsequent tests was therefore subject to slight variations as the "refilled-crater" was not perfectly leveled. Visual alignment of explosive and target was employed, as a transit was not available for these later experiments.

Another possible error source lies with the initiation techniques. A No. 8 blasting cap was centrally located on the flat side of the hemispherical charge. On several instances however the blasting cap was placed on the surface of the charge. The depth to which the blasting cap was inserted was likewise not closely controlled.

All of the planned future experiments are designed to benefit from the shortcomings and experience gained in the earlier experiments. Cast composition B hemispheres with machined cavities to accommodate the blasting cap have been prepared. It is expected that a surveyors transit will be available so as to insure proper alignment. Several new pressure gages are under consideration for incorporation in later tests. The resonant frequency of these gages are believed to be high enough so as to eliminate the "ringing" noted in the present gage records. As human error is eliminated and repeatability is obtained, experimental tolerance bands will be computed based on the logarithmic derivative method. Subsequent tests should indicate whether this "gage ringing" is a function of the gage or the system. The natural frequencies of the target will be calculated (Ref. 3) and compared with the "ringing" frequency noted in the reflected pressure records. Dr. Emmet Wittmer of Massachusetts Institute of Technology earlier had suggested that a diameter to thickness ratio of 60 to one for our target is too low to excite any but the first vibrational mode. This will be investigated further.

Also under consideration for future tests is the use of a 7 channel tape recorder. This would enable us to instrument the models more fully. Based on our discussions with the personnel of Suffield Experimental Station, methods are being studied by our instrumentation group to improve their field calibration technique.

The present experiments at Picatinny involve the use of one target model mounted on a stand 15 feet from the explosive charge. It is felt that this represents a high inertia type system. One of the experiments planned for the future is to freely suspend an instrumented target and subject it to the same level of blast as the rigidly mounted targets. The records will then be compared

so as to assess the major differences and to explain whether or not a ground coupled shock exists and is distorting the records. The majority of the experiments currently being conducted at the Picatinny Arsenal test range are with Target No. 2 at free air overpressure levels of 25 psi and 75 psi. Strain data at the 75 psi overpressure level has not been recorded as yet but this will be incorporated into the next series of experiments. It is interesting to note that the duration of the strain information appears in good agreement with the duration of reflected pressure measurements. When plotted on a common axis, the strain time histories (See Fig 8-11) all follow similar shapes. The figures are not true representations of the actual strain time histories. They were drawn by simply connecting maximum and minimum points from the original traces. They are placed here merely as an indication of the reproducibility of the strain data as to general shape. The amplitudes vary somewhat due to the variations in reflected pressure, resulting from the experimental errors discussed previously. After the first few cycles the strain traces become rather spurious. The maximum strain recorded for an overpressure of 25 psig was 183 micro inches per inch. For an aluminum shell having a modulus of elasticity of  $10^7$  psi, the 25 psig overpressure leads to a stress of 1830 psi. Based on the experimental data of W. Schuman of BRL, target No. 2 should fail at an overpressure of 167 psi from 10 pounds of explosive. As future experiments are carried out it is expected that the stress at which a target buckles will be determined. For hard targets, buckling may not be a principal criteria for damage and the determination of the experimental stress level at which yielding occurs could prove valuable. Figure 12 is an actual oscilloscope record of the strain time history for test SP-3.

#### Planned Tests at Off-Arsenal Site

It is presently planned that several large scale tests will be conducted at a proving ground during the spring or summer of 1963. Inquiries were sent to White Sands Proving Grounds, White Sands, New Mexico; Yuma Test Station, Yuma, Arizona, and Naval Ordnance Test Station, China Lake, California. The replies were considered, with respect to cost and facilities available, and it was determined that Yuma or White Sands would be most suitable for a planned test purpose.

At White Sands there is available a 17 channel, high frequency tape recorder set up in a trailer, blast pressure gages and a 16 channel cathode ray tube high frequency recording unit. In the past, White Sands has successfully detonated multi-ton charges of H. E. They will build the hemispherical charge, supply photo coverage, supply trailers and vans for our equipment, and aid in data reduction.

Yuma Test Station has on hand 8 channels of oscilloscope equipment and 13 channels of high frequency tape recorders. They could also supply another 4 channels of high frequency recording equipment if Picatinny Arsenal was to provide the cathode followers for the gages. At Yuma, due to their summer environmental test programs, the possibility of delays is present.

A final decision as to which facility will be used will be made shortly in coordination with BRL.

The tests to take place at a proving ground site are presently planned to use one-ton and five-tons of TNT for two separate experiments. Picatinny Arsenal will supply four 4-beam oscilloscopes to supplement the instrumentation at the proving ground facility. A total of forty channels of information would then be recorded during each test. Four models of various geometric parameters will be instrumented with 10 channels of information on each model. Four channels of blast pressure (face on and  $45^{\circ}$ ), four channels of strain (face on and  $90^{\circ}$ ), and two channels of free air information will be obtained from each model. The use of a strain gage having a transverse sensitivity coefficient such that stress may be read directly, is presently being investigated (Reference 13). This gage not only reads stress directly but also permits the use of only one channel of information to measure strain in two directions. If more channels of high frequency information become available, additional models will be instrumented.

Also proposed for the large scale H. E. tests would be the emplacement of approximately 20 uninstrumented models on test stands subjected to a variety of pressure levels. At present it has been planned to have 4 different models available for tests. These will be cone-cylinder combinations with hemispherical end caps. The cylindrical portions of the models would have the following dimensions.

Target 1 - 7-1/2" dia. cylinder x 20" long x 1/16" wall thickness

Target 2 - 7-1/2" dia. cylinder x 20" long x 1/8" wall thickness

Target 3 - 10" dia. cylinder x 20" long x 1/16" wall thickness

Target 4 - 10" dia. cylinder x 20" long x 1/8" wall thickness

(See Figures 2, 3, and 4 for typical target configurations). The models are fabricated of 6061 T6 aluminum. These have diameter to thickness ratios of 120, 60, 160, and 80, respectively. These models will be visually inspected and photographed to ascertain any damage. The models will encompass an overpressure region from approximately

3 psi to a region approximately 3 times greater than that predicted for incipient buckling by Schumann's work pertaining to our specific models (See table 3 in later section of this report). Attempts would then follow to determine "kill" criteria based on the damage levels of the uninstrumented models. Free field pressure gages will be placed at the uninstrumented model locations if additional high frequency recording equipment is available. Otherwise, durations, overpressures and impulses will have to be scaled from TNT and pentolite experimental data.

The four instrumented targets will be placed in overpressure regions somewhat less than the predicted incipient buckling pressures of Table 3. It is necessary to keep the instrumented targets in the elastic response range, and at a pressure level somewhat below the predicted incipient buckling pressures for the targets, as the strain gages are limited not by the modulus of elasticity of the target but by the modulus of elasticity of the bonding material which fixes the strain gage to the target. The response of the models at these overpressure levels will be recorded through the strain gages and will then be correlated with the tests at Picatinny Arsenal at similar overpressure levels but at lower durations and consequently at lower positive impulses.

Table 2 below indicates the scaled durations and impulses of interest for the overpressures noted, (Reference 4, 5, 6).

TABLE 2

Charge weight (w/ Reflection Factor = 1.8) # TNT	Scaled Range	Over Pressure	Duration msec	Side on Positive Impulse (lb msec/in <sup>2</sup> )
20.7 lbs.*	5.44	25 psig	3.0	30.3
103.5 lbs.*	3.20	75 psig	2.35	45.2
3,600 lbs.	5.44	25 psig	17	170
3,600 lbs.	3.20	75 psig	7.7	216
18,000 lbs.	5.44	25 psig	29	290
18,000 lbs.	3.20	75 psig	13.15	368

\* These tests have been performed at Picatinny Arsenal and will be repeated with the intent of minimizing experimental error.



Table 3 indicates the expected dynamic buckling pressures for the targets and weight of explosive noted and tabulates these alongside the computed static buckling pressures.

TABLE 3

Predicted Buckling Pressures of Cylindrical Portion of Targets

Target	Dynamic Buckling Pressure *a				Static Buckling Pressure	Reflected Pressure Factor
	2000 lbs. TNT		10,000 lbs. TNT			
	*b	*c	*b	*c		
1	37 psig	32.1 psig	34 psig	31.0 psig	150 psig	2.5
2	94	110	87	100	850 "	4.5
3	24.5	13.2	24	13	40 "	2.0
4	65.5	49.6	61.5	46	216 "	2.5

\*a - In terms of free-air overpressure for the explosive noted.

\*b - These pressure levels are predictions based on the experimental data of W. Schuman of Ballistic Research Labs. (Reference 11)

\*c - These pressure levels are calculated based on Picatinny Arsenal Report No. 61-EA-28 by Carl Larson (Reference 7). (See Appendix)

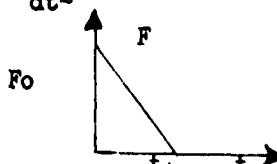
Two techniques have been employed to predict the incipient buckling pressures. Good agreement is observed for both techniques for targets 1, 2, and 4. Target 3 however does not enjoy this proximity. Since Schuman's work is based on experiment it might be reasonable to assume that the theory used to predict the incipient buckling pressures based on triangular pulse loading is in need of some correction in the low pressure region. The determination of the incipient buckling pressure based on the triangular pulse loading is outlined in Appendix 1. Briefly stated, the approach is to compute "dynamic load factors" for the targets based on a triangular type of impulse loading and assume that the deflection of the fundamental mode is approximately equal to the deflection of an equivalent static load. The target is analyzed as a simple system with a single degree of freedom. This is true provided that the duration of impact is sufficiently long (a tenth or more) of the fundamental natural period of the target. (Reference 14) In the experiments at Picatinny Arsenal, Southwest Research Corporation (Reference 8), and Suffield Experimental Station (Reference 9), it has been found that the reflected pressure on a cylindrical model, of approximately the size of the targets used in the Picatinny Arsenal tests, is 2.5 to 3 times as great as the overpressure. The

experiments at all three installations were carried out at approximately 25 psi overpressure. In the computation of "dynamic load factors", a factor of 2.5 was used as the ratio of face-on reflected pressure of the cylindrical shell to the free air overpressure for targets 1 and 4. Curves such as figures 3 and 4 of BRL Report No. 988 (Reference 4) indicate that in the higher impulse and pressure regions a factor of from 4 to 8 for the ratio of face-on pressure (impulse) to side-on pressure (impulse) is more likely. It can be shown (Reference 10) that a maximum factor of 8 can exist for strong shock waves as a ratio of face-on pressure to side-on pressure. Therefore it seems reasonable to use a factor greater than 2.5 for target No. 2, so that better correlation exists between the two techniques shown in Table 3 (e.g. for a factor of 4.5 the predicted buckling pressure is approximately 100 psi for target #2). Likewise at low pressures and impulses, a reflected pressure factor of 2.5 is too great for the ratio of face-on to side-on pressure (e.g. for a factor of 2.0 for target #3 the predicted buckling pressure is approximately 13 psi). As further experiments are carried out at Picatinny Arsenal, Ballistic Research Labs and Suffield Experimental Station, the pressure distribution around a circular shell for various shell geometries and shock strengths will be determined. These corrections will then be incorporated into future theoretical analyses.

## APPENDIX I

### Determination of Dynamic Load Factor Based on Triangular Type of Impulse Loading (Reference 7)

The differential equation for the triangular impulse loading for a simple one degree of freedom system is,

$$m \frac{d^2x}{dt^2} + kx = F_0 - F_0 \left( \frac{t}{t_1} \right)$$


Larson then solves this differential equation, noting that

$$w^2 = \frac{k}{m} \quad \text{where} \quad w = \sqrt{\frac{gp}{yc}}$$

The solution is

$$x = \frac{F_0}{kwt_1} \left[ \sin(wt) - wt, \cos(wt) - \cos w(t-t_1) \right] \quad c \leq t \leq t_1$$

$$x = \frac{F_0}{kwt_1} \left[ \sin(wt) - wt, \cos(wt) - \sin w(t-t_1) \right] \quad t \geq t_1$$

where  $F$  = concentrated load (lbs);

$F_0$  = peak concentrated load (lbs)

$t$  = time (sec)

$t_1$  = duration of pulse (sec)

$k$  = spring constant (lbs/in.)

$m$  = mass (lbs-sec<sup>2</sup>/in.)

$x$  = amplitude of deflection (in)

$g$  = acceleration due to gravity (in/sec<sup>2</sup>)

$y$  = maximum static deflection of structural element (in)

$p$  = load (lbs/in)

$c$  = weight of structural element per unit length (lbs/in)

and  $A$  = Dynamic load factor

$W$  = Weight of explosive (lbs)

$Z$  = Scaled distance (feet/lbs  $^{1/3}$ )

} used on p. 15

The general equation for a spring force  $S$  for  $t \leq t_f$  is:

$$S = kx = F_0 \left[ 1 - \frac{t}{t_f} - \cos \omega t + \frac{\sin \omega t}{\omega t_f} \right]$$

The ratio of the spring force  $S$  to the maximum dynamic load  $F_0$  will be defined as the dynamic load factor and noted as  $A$  (Reference 14). If a target buckles statically at  $P_b$ , then under dynamic conditions which lead to dynamic load factor  $A$ , the target will buckle at  $P_b/A$ . The maximum dynamic load factor when the peak amplitude occurs for  $0 < t < t_f$  is:

$$A = 1 - \frac{\omega t}{\omega t_f} - \cos \omega t + \frac{\sin \omega t}{\omega t_f}$$

A sample computation of the dynamic load factor for target No. 1, in a test using 10,000 lbs. of TNT placed on the ground in an hemisphere so as to obtain a reflection factor of 1.8, follows.

The static buckling pressure computed for the cylindrical portion of target No. 1 is computed from the relation, (Reference 12).

$$P_b = \frac{.365 E}{\frac{L}{R} \left( \frac{R}{h} \right)^{2.5}} \quad \text{For a material with a Poisson's Ratio} = 0.3$$

where;

$P_b$  = critical buckling pressure, (psi)

$E$  = youngs modulus, (psi)

$h$  = shell thickness, (in.)

$L$  = unsupported length of cylinder, (in.)

$R$  = radius of cylinder, (in.)

For Target No. 1  $P_b = 150$  psi

Using a factor of 2.5 for the ratio of face-on reflected pressure of the cylinder to the free-air overpressure, an overpressure of

$$\Delta P = \frac{150}{2.5} = 60 \text{ psig is used in the analysis.}$$

The scaled distance from the center of charge (10,000 lbs) where 60 psi overpressure will be experienced is

$$Z = 3.9 \frac{\text{ft}}{\text{lbs}}^{1/3} \quad (\text{Reference 4, 5, 6})$$

$$Z = \frac{\text{Distance from charge (R)}}{(\text{Explosive Weight})^{1/3}} \quad (\text{lbs})^{1/3}$$

$$R = (10,000 \times 1.8)^{1/3} (3.9) = 102.5 \text{ feet.}$$

For a scaled distance of 3.9 (Reference 4)

$$\frac{t_d}{W^{1/3}} = .7$$

or the length of the pulse,  $t_p = 18.4 \text{ msec.}$

The maximum static load deflection from Roark, (Reference 12), Table XIII, Case 12 is

$$y = (.0136) (10^{-6}) R^{3/4} L^{3/2} h^{9/4} p$$

The weight of the aluminum target is

$$c = (.100) \pi D h = .628 Rh$$

Then,

$$wt_1 = \left( \sqrt{\frac{EP}{yc}} \right) t_1 = \left[ \frac{386 \times 10^6 \times h^{9/4}}{.0136 \times R^{3/4} L^{3/2} \times .628 Rh} \right]^{1/2}$$

$$wt_1 = 2.13 \times 10^5 \frac{h^{5/8} t_1}{R^{7/8} L^{3/4}}$$

For target No. 1

$$wt_1 = 47.8$$

The number of radians to the point of maximum amplitude is:

$$wt_1 \sin wt + \cos wt = 1$$

$$wt = 3.125$$

The maximum dynamic load factor:

$$A = 1 - \frac{wt}{wt_1} - \cos wt + \frac{\sin wt}{wt_1}$$

$$A = 1.934$$

Therefore under the conditions cited (i.e. 102.5 feet from 10,000 lbs of TNT w/Reflection factor = 1.8) the target should buckle at an overpressure of

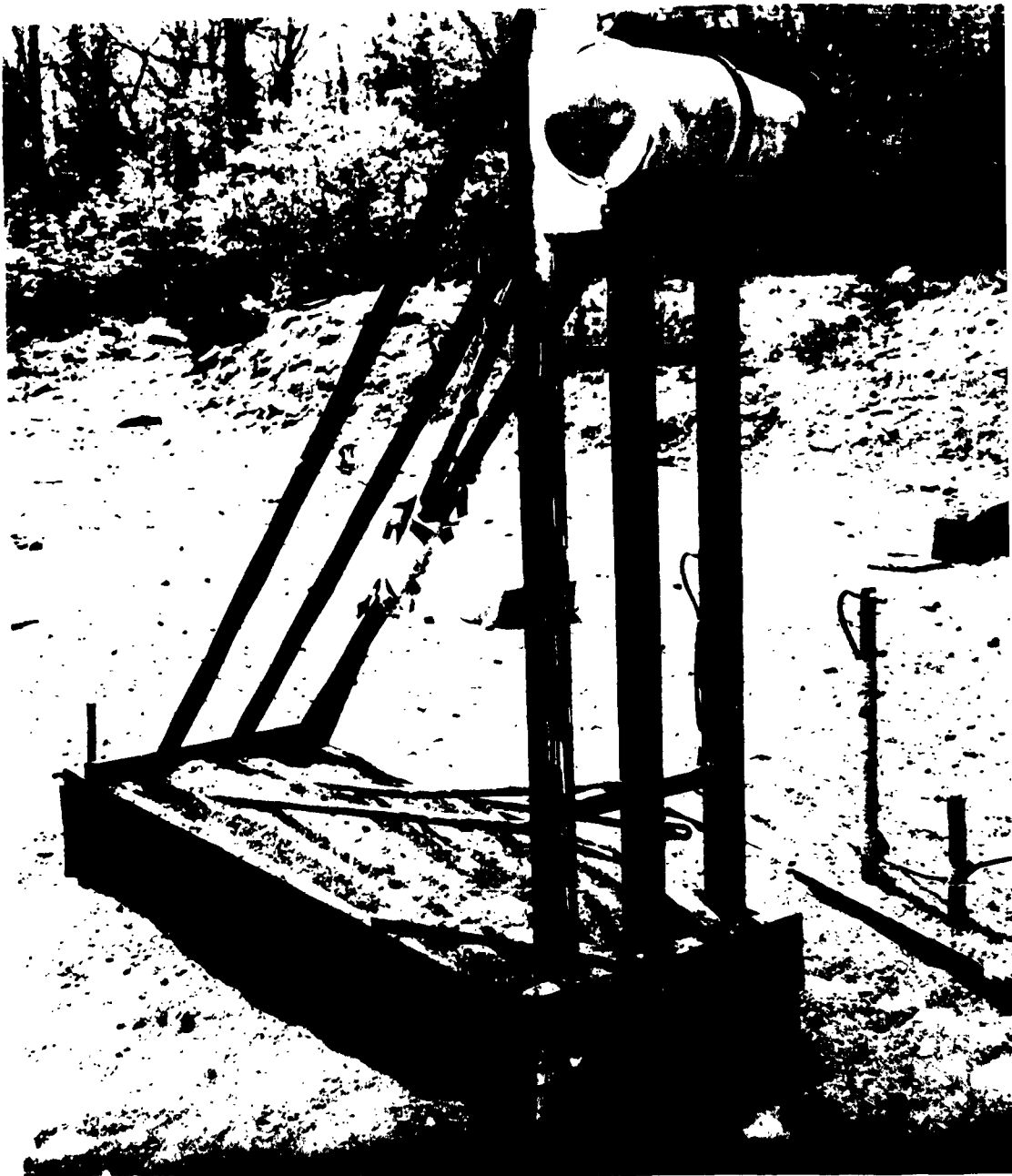
$$\Delta P = \frac{60}{1.934} = 31 \text{ psig.}$$

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28-017-656/Ord-62  
Figure 2

PICATINNY ARSENAL

ORDNANCE CORPS

Steel Stand W/Wooden Sand-Filled-Box-Base

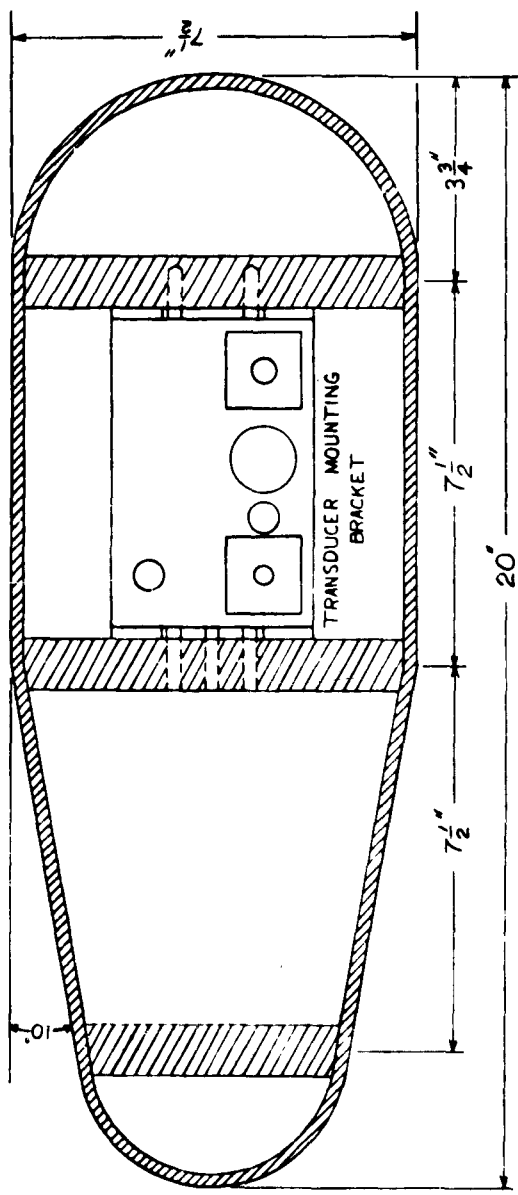


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SYN	DESCRIPTION	DATE	APPROVAL

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ORDNANCE PART NO.

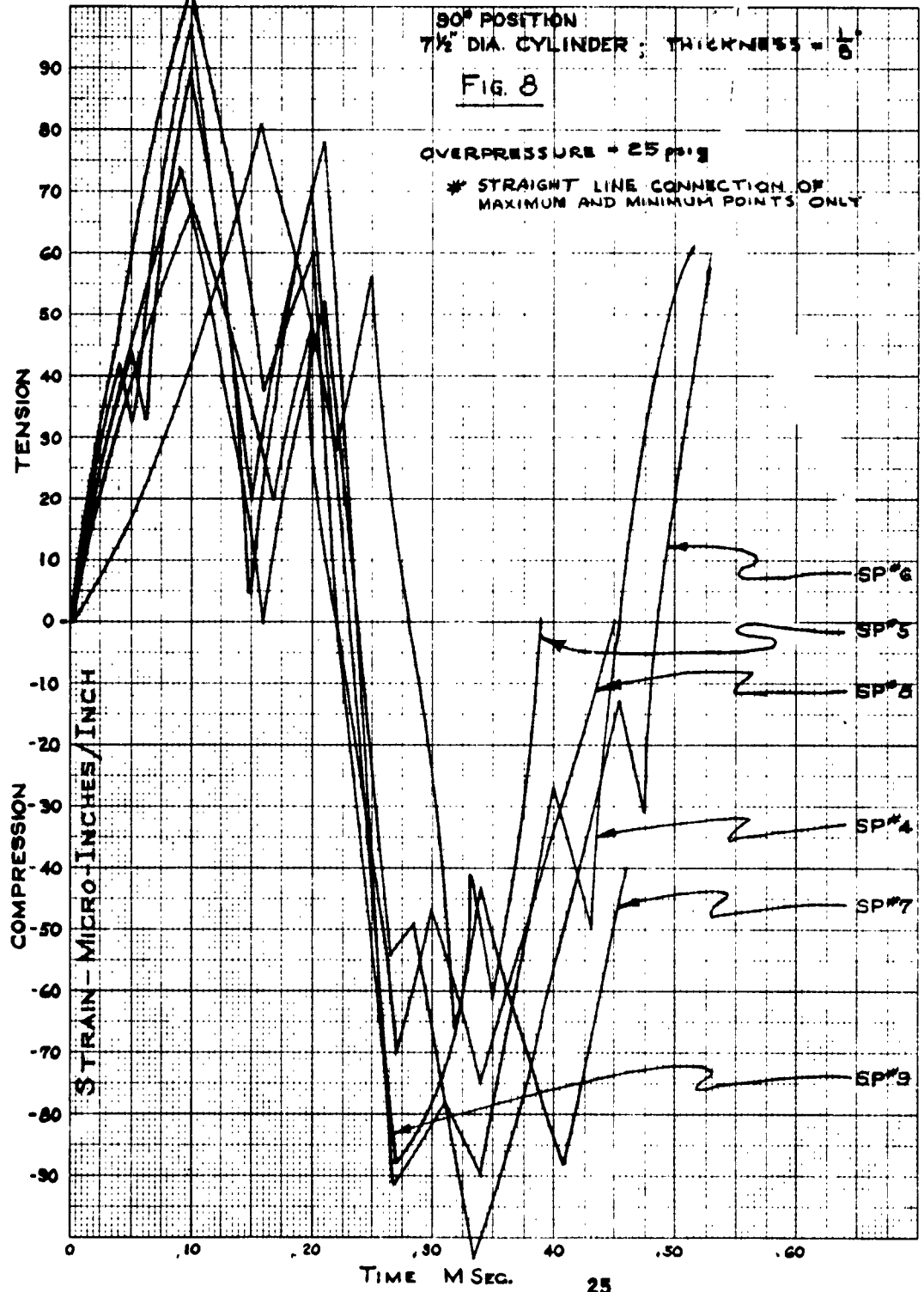
PICATINNY ARSENAL ORDNANCE CORPS DEPT OF THE ARMY DOVER, NEW JERSEY		Figure 5 GEOMETRIC TEST MODEL		SHEET B	OF
ORIGINAL DATE OF DRAWING		UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES TOLERANCES ON FRACTIONS DECIMALS ANGLES		APPROVED BY CHIEF OF ORDNANCE	
DIMENSIONS TOLERANCES DECIMALS ANGLES		MATERIAL 6061 T-6 ALUMINUM		APPROVED BY CHIEF OF ORDNANCE	
FINISH RAZED		HEAT TREATMENT		APPROVED BY CHIEF OF ORDNANCE	
FINISH RAZED		FINAL PROTECTIVE FINISH		APPROVED BY CHIEF OF ORDNANCE	
PHYSICAL PROPERTIES		TYPICAL TYPICAL TYPICAL		PHYSICAL PROPERTIES	
SEE ENGINEERING RECORDS NEXT PART		USED ON		USED ON	
APPLICATION		APPLY PART NO.		AS SPECIFIED	
DO NOT DO		DO NOT DO		DO NOT DO	



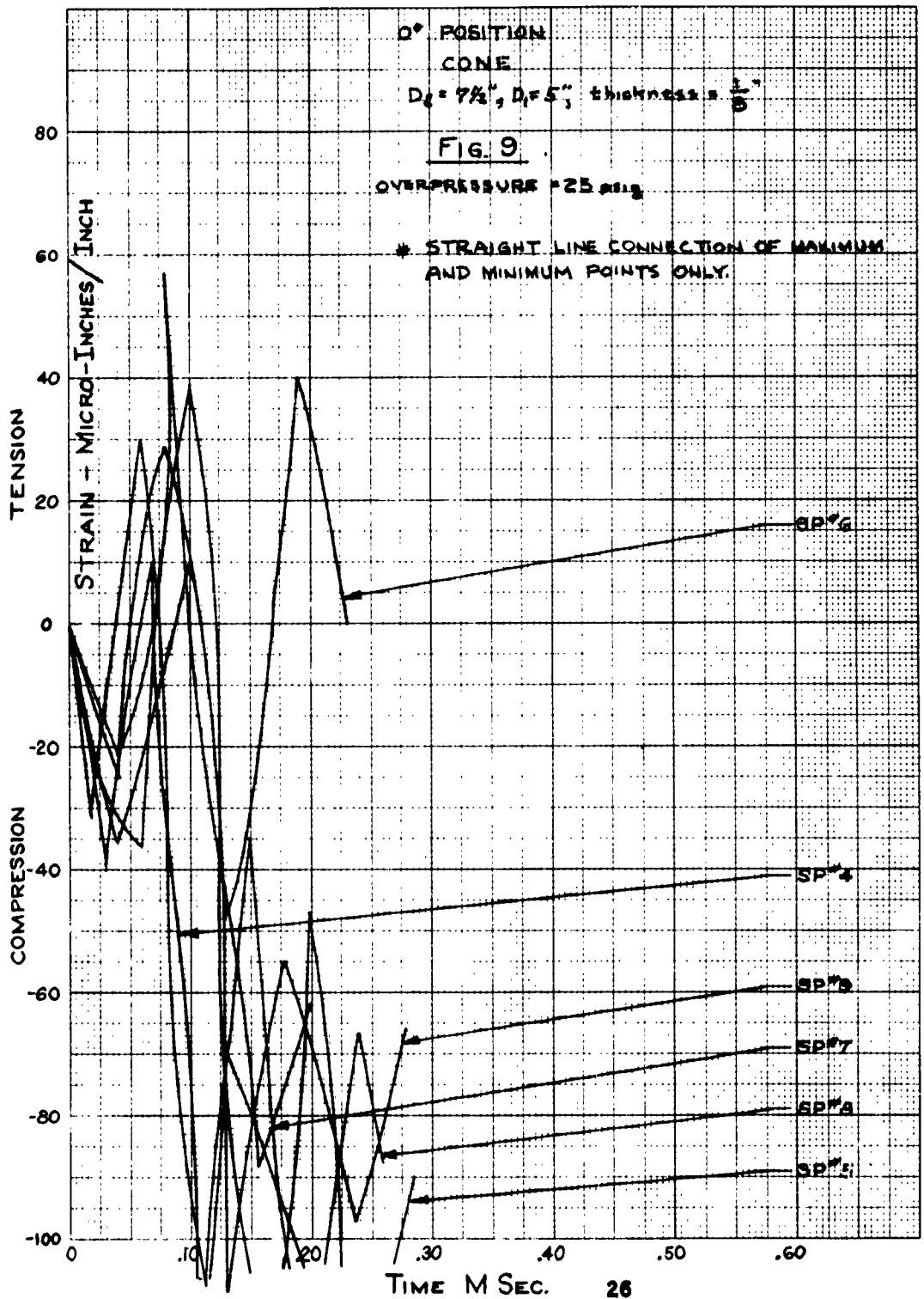
23



# STRAIN VS. TIME \*



# STRAIN VS. TIME \*





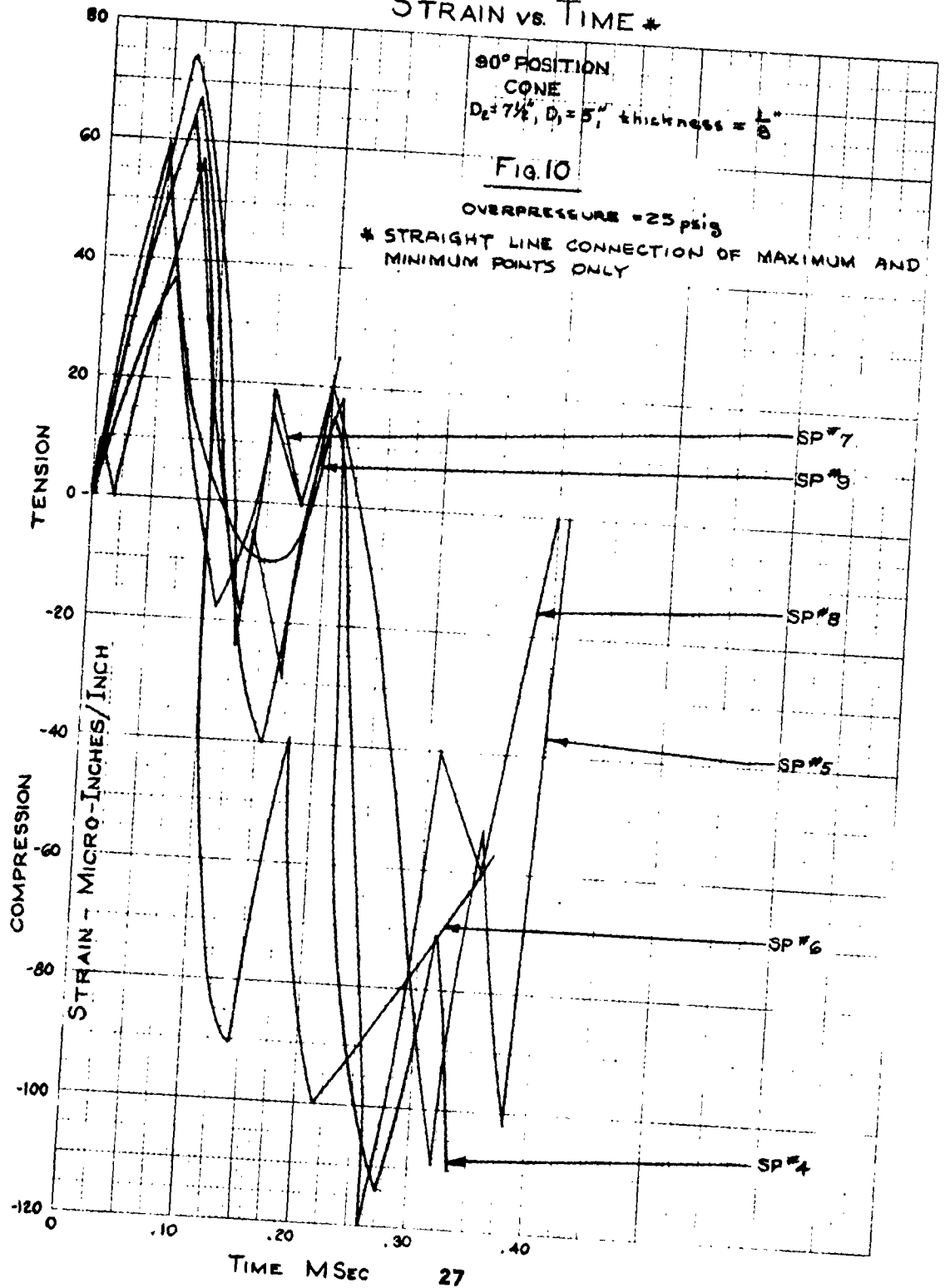
# STRAIN vs. TIME \*

90° POSITION  
CONE  
 $D_2 = 7\frac{1}{4}"$ ,  $D_1 = 5"$ , thickness =  $\frac{1}{8}"$

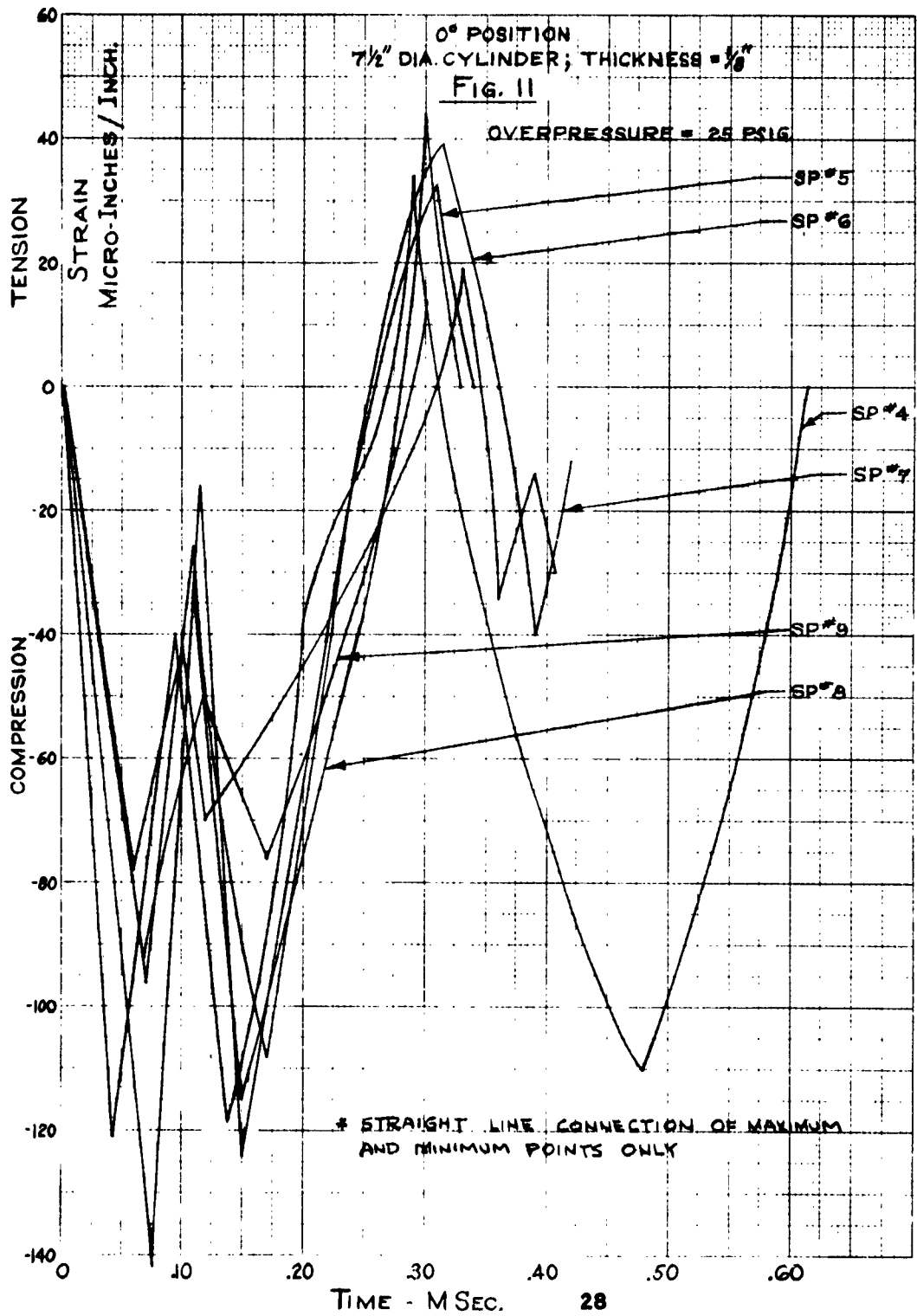
Fig. 10

OVERPRESSURE = 25 psig

\* STRAIGHT LINE CONNECTION OF MAXIMUM AND MINIMUM POINTS ONLY



# STRAIN VS. TIME \*



Firing pulse

Time of arrival  
14 feet

Time of arrival  
16 feet

2138

29

Target no. 2

Time of arrival, 2 foot interval

Pressure, 0 degrees, cylinder

Pressure, 45 degrees, cylinder

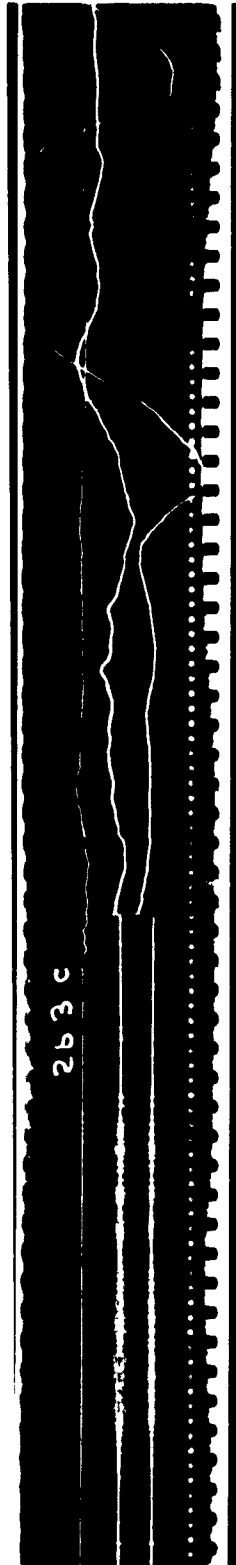
Timing, 10 sec

Figure 12

Firing pulse

Time of arrival  
@ 14 feet

Time of arrival  
@ 16 feet



30

TEST CP 3, Target No. 2

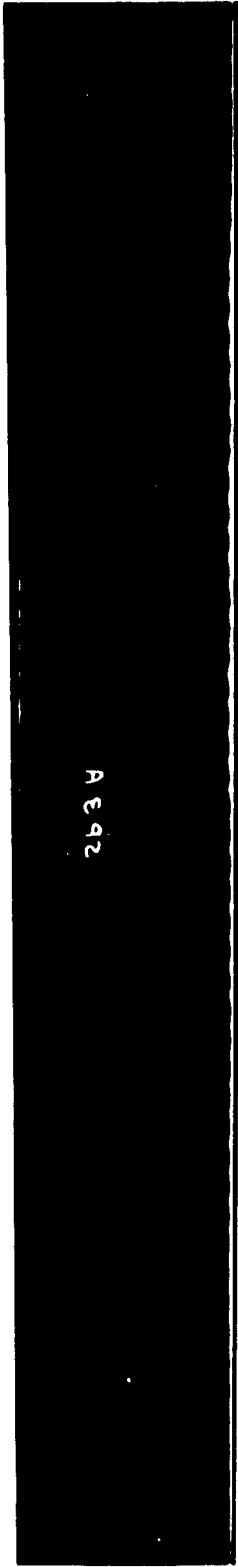
Time of arrival, 2 foot interval

Pressure, 45 degrees, cone

Pressure, 0 degrees, cone

Timing, 10 MC

Figure 13



2b3 A

3, Target No. 2  
train, 90 degrees, cylinder  
train, 90 degrees, cone  
train, 0 degrees, cone  
train, 0 degrees, cylinder  
Timing, 10 sec  
figure 14

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